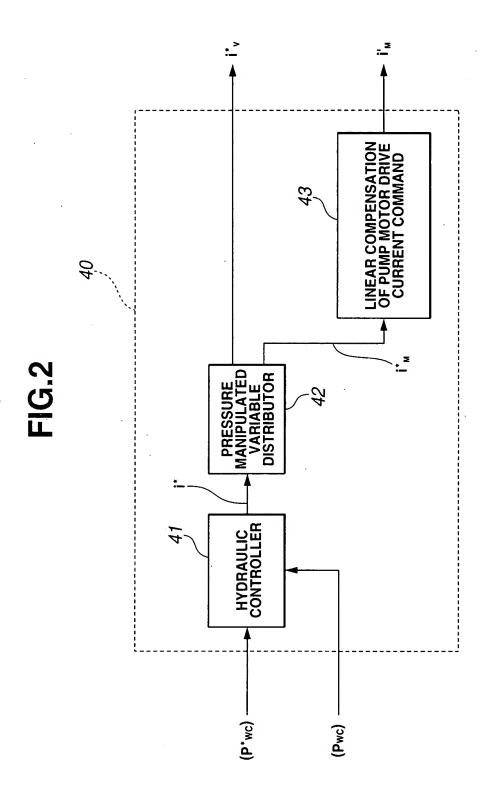


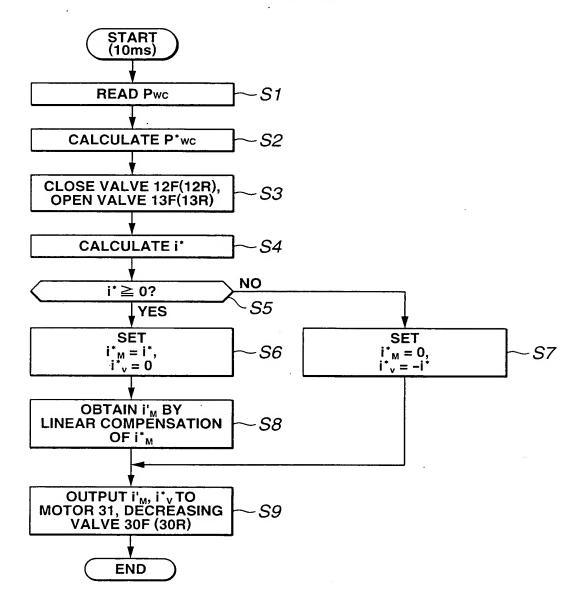
Title: ELECTRONICALLY
CONTROLLED HYDRAULIC BRAKE
SYSTEM
Inventor(s): Jicheng ZHANG et al.
DOCKET NO.: 023971-0387

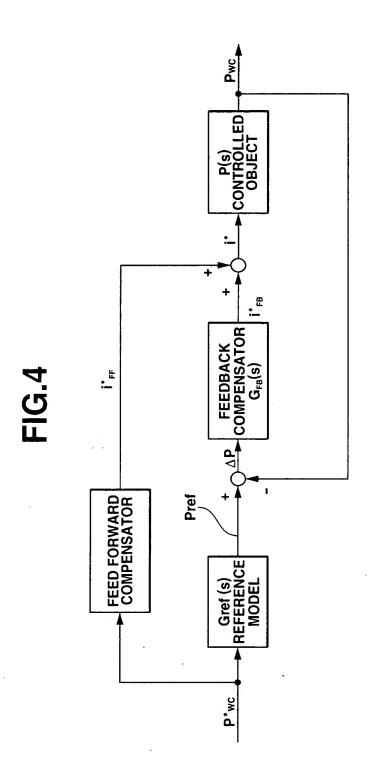


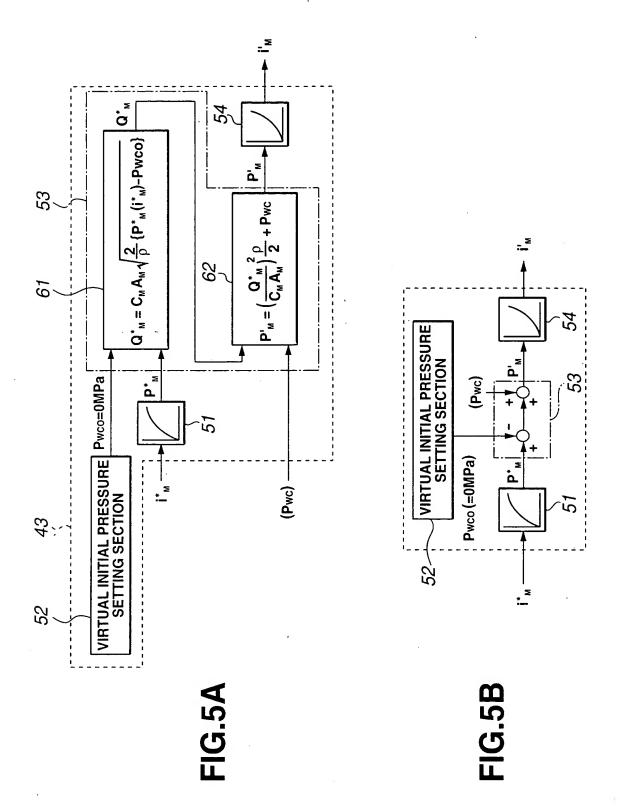
Title: ELECTRONICALLY CONTROLLED HYDRAULIC BRAKE SYSTEM

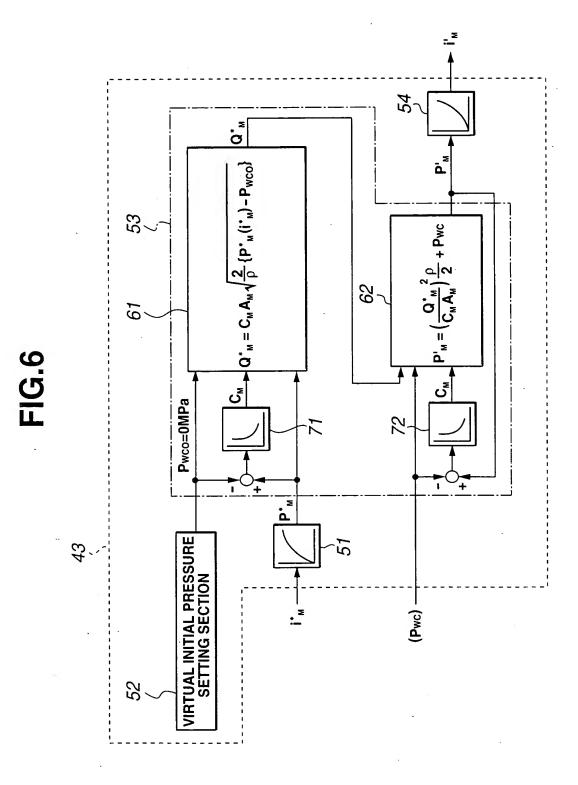
Inventor(s): Jicheng ZHANG et al. DOCKET NO.: 023971-0387

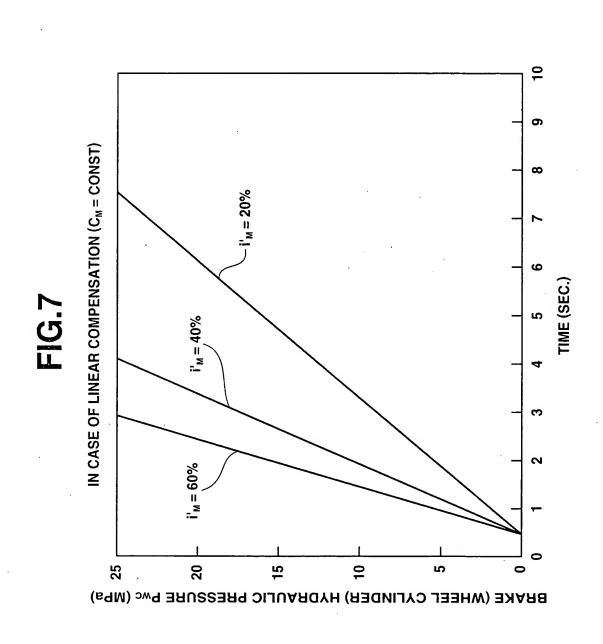
FIG.3



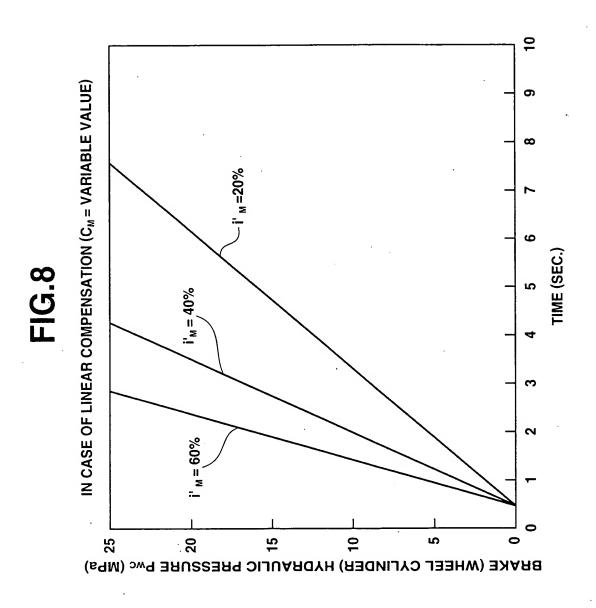




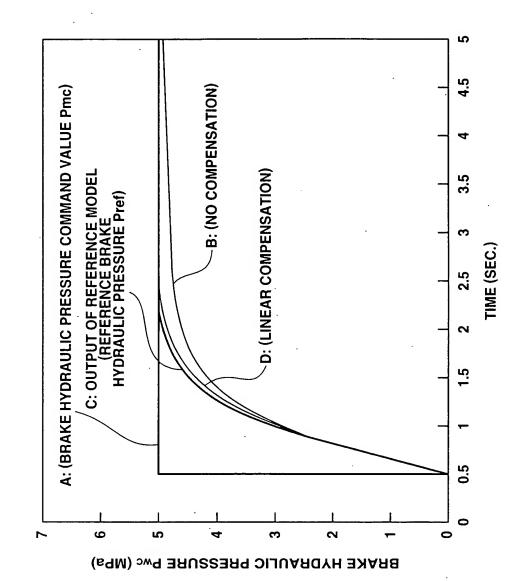


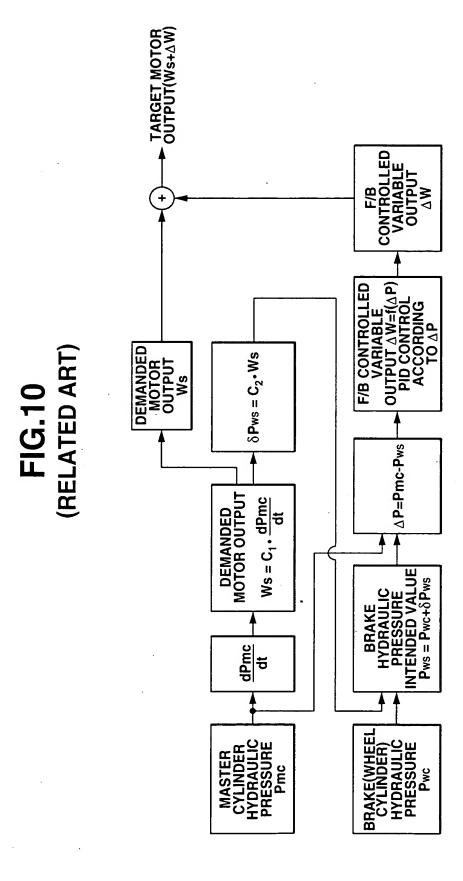


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-16.9

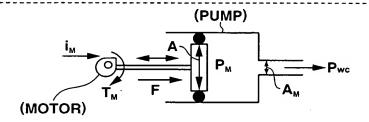




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FIG.11



im: PUMP DRIVE MOTOR CURRENT [A] (PROPORTION TO

T_M: PUMP DRIVE MOTOR AXIAL TORQUE [N·s]

F: FORCE APPLIED TO CROSS-SECTION A [N]

A: PISTON CROSS SECTIONAL AREA [m²]

P_M: ORIFICE UP STREAM (PROPORTION TO PISTON INNER PRESSURE, PUMP DRIVE PRESSURE [MPa]

MOTOR CURRENT)

Pwc: ORIFICE DOWN STREAM PRESSURE [MPa] (HYDRAULIC PRESSURE)

A_M: PUMP ORIFICE OPENING CROSS-SECTION [m²]

Q_M: PUMP VOLUMETRIC FLOW RATE [m³/s]

C_M: PRESSURE INCREASING SIDE FLOW RATE COEFFICIENT

Q_M: FLUID DENSITY [kg/m³]

PUMP FLOW RATE MODEL:

$$Q_{\text{M}} = C_{\text{M}} A_{\text{M}} \sqrt{\frac{2}{\rho} \left\{ P_{\text{M}}(i_{\text{M}}) - P_{\text{WC}} \right\}}$$

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FIG.12 (RELATED ART)

